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**GUGGENHEIM AERONAUTICAL LABORATORY
CALIFORNIA INSTITUTE OF TECHNOLOGY**

HYPersonic RESEARCH PROJECT

Memorandum No. 57

July 20, 1960

**A STUDY OF AREA CHANGE
NEAR THE DIAPHRAGM OF A SHOCK TUBE**

by

David A. Russell



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ABSTRACT

The performance of a shock tube with area change near the diaphragm has been calculated for the high shock speed case by previous authors. This report extends the ideal theory calculations to include the whole shock speed range and all possible area changes near the diaphragm. Simple calculation procedures are presented and the practical applications of this type of area change are discussed.

Experimental measurements show excellent agreement with the theory except for the intermediate shock speed range where a non-steady secondary shock wave is predicted. Here not only is the agreement with the basic performance curves marginal, but detailed observations revealed that no secondary normal shock is present! A new model is devised to explain these discrepancies. This new model takes frictional effects into account and shows promise of being useful for other shock tube problems.

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LIST OF SYMBOLS

a	speed of sound
A	cross-sectional area
A^*	throat cross-sectional area ($M = 1$)
A_s	cross-sectional area at shock position in nozzle (Appendix I. C)
g	amplification factor (Appendix I. A., Eq. 9)
g_s	amplification factor for shock-in-nozzle configuration (Appendix I. C)
M	Mach number
M_s	primary shock Mach number
P	static pressure
P_{Tg}	total pressure after probe normal shock wave
s	slope of fine wire response
t	ideal theory calculated time of arrival of event (measured from passage of primary shock wave)
	t_{cs} contact surface arrival
	t_{s2} secondary shock arrival
	t_e secondary expansion wave arrival
	t_{ee} end of secondary expansion wave
T	static temperature
u	velocity
u_{s2}	speed of secondary shock wave down the shock tube
x	distance along the shock tube (measured from the diaphragm station)
γ	ratio of specific heats
ρ	density

Subscripts

$()_1$	parameter in region 1, the undisturbed driven gas
$()_2$	parameter in region 2, downstream of contact surface
$()_3$	parameter in region 3, upstream of contact surface
$()_4$	parameter in region 4, the undisturbed driver gas

$\left. \begin{array}{l} ()_{3'} \\ ()_{3''} \\ ()_{4'} \end{array} \right\}$ parameters in or adjacent to the area change (See Figure 1.)

$()_a$ parameter in a general region a

$()_b$ parameter in a general region b

I. INTRODUCTION

The types of cross-sectional area change that affect wave motion in a duct may be conveniently classified when boundary layer effects can be neglected. First there is the group in which, as a result of the area change, accelerating or decelerating waves are an essential feature of the flow (i. e., a long tapered tube), and second, the group for which all waves may be treated as having no acceleration. This classification is based on the relative length of time that the wave system spends in the region of varying area; the first group corresponds to the case where the wave system never emerges from the region of varying area, while in the second group the wave system has long since passed through the area change.

The configurations involving no wave acceleration have similarity solutions in the x - t plane about some origin. They may be divided into those configurations for which there are waves entering as well as leaving this origin (i. e., problems involving wave reflection¹)*, and those for which there are only waves leaving the similarity origin. For a shock tube the latter group is referred to as having area change "near the diaphragm". This implies that, for flow times of interest, the various waves have attained a state of constant velocity, wave reflection has ceased to be important, and the contact surface has passed through the region of varying area. The purpose of this report is to study this type of area change in a shock tube.

Shock tubes with area change near the diaphragm may be treated in ideal theory (Cf. Section II.) as though the area change were an isentropic nozzle located at the diaphragm. The calculation of shock tube performance is then basically a problem of matching pressure and velocity across the contact surface. In general, an upstream-facing secondary wave, between the nozzle exit and the contact surface, will be found necessary in order to achieve the contact surface matching. (Cf. Figure 1) For very high shock Mach numbers, this wave will be a non-steady expansion wave; for lower Mach numbers, a non-steady normal shock

* Superscripts denote references at the end of the text.

wave is required; for still lower Mach numbers, the normal shock wave moves into the nozzle and becomes stationary; while at the very low Mach numbers, the nozzle flow becomes subsonic, and thus incapable of supporting any secondary wave.

The above wave configurations are the only possible ones for a shock tube with area change near the diaphragm; however, a given shock tube may not produce them all. Previous work, notably by Resler, Lin, and Kantrowitz² and by Alpher and White³*, has been concerned with the use of area change at the diaphragm for the attainment of high speed shocks, and thus has only partially considered the first configuration (the case where the secondary wave is isentropic). In this report, the existing work is extended to all four configurations, and thus covers the whole range of shock Mach numbers and possible area changes at the diaphragm.

The ideal theory for the different configurations is discussed in the first section of the report. (Detailed calculation procedures are presented in Appendix I.) Curves of range of application and of initial pressure ratio versus shock Mach number are presented for Nitrogen-Air and Helium-Air shock tubes. The second section discusses the usefulness of various area configurations with the aid of the ideal theory. Particular attention is given to the use of a simple drilled plate inserted at the diaphragm, as a means of gaining additional flexibility in shock tube operation.

The agreement between experimental and theoretical curves of initial pressure ratio versus shock Mach number is discussed in the next section, where the fine wire and piezo-electric pressure gauge observations of the secondary waves are shown to indicate the necessity for a new model for the non-steady, non-isentropic configuration.

A simple model, based on the concept of the Fanno process, or spread out compression region, is presented in the fourth section. With this new theory the remarkable agreement with experimental basic performance is shown, and the insensitivity of the basic performance curves to the type of model is demonstrated. The inadequacies of this new theory are discussed, and more realistic models are indicated.

* Reference 3 includes a critical review of earlier work, and in particular points out an error in a previous study of the expansion wave configuration made at this laboratory. (Yoler, Y. A.: Hypersonic Shock Tube. GALCIT Hypersonic Research Project, Memorandum No. 18, July 19, 1954.)

II. IDEAL THEORY

The ideal theory is based on the assumptions of no heat transfer and no boundary layers or associated viscous interaction regions. These restrictive assumptions, which allow the use of isentropic nozzles and simple plane waves, will turn out to be remarkably good for predicting most of the performance characteristics of shock tubes with area change near the diaphragm. The procedure is based on matching conditions across the contact surface, and is set up so as to avoid the iterative calculations that are common to this type of problem.

The shock tube configurations studied are illustrated in Figure 1. Each model shown is similar to a plain shock tube, except for the changing area at the diaphragm station and the resultant secondary waves. The notation used is adapted from the accepted shock tube notation, where region 1 refers to the undisturbed driven gas, regions 2 and 3 to the gas downstream and upstream of the contact surface, and region 4 to the undisturbed driver gas. The primed numbers refer to the gas immediately upstream and downstream of the area change as shown. It should be noted that the secondary waves are always upstream of the contact surface, and are always upstream facing waves (although nonsteady waves are swept downstream by the flow). It can be shown that any other secondary wave configuration is unstable, for the wave will either tend to change its position to correspond to one of the configurations in Figure 1, or it will catch up with one of the primary waves.

The assumption that the area change may be treated as an isentropic nozzle enables the shock tube geometry to be completely specified by A_4/A_1 (the ratio of driver to driven tube area) and A^*/A_1 (the ratio of the throat area to the driven tube area). For a given shock tube with specified gases (i. e., a_4 , γ_4 , a_1 , γ_1), the velocity downstream of the contact surface is uniquely determined by M_s (the shock Mach number), while the velocity upstream of the contact surface is fixed by the nozzle geometry (if the nozzle exit flow is assumed supersonic). Thus, there is only one value of M_s for which the velocity will match across the contact surface. For shock speeds above this value a second expansion wave is necessary to further expand the nozzle exit flow. This is the "expansion wave configuration" depicted in Figure 1a. For shock speeds below the

critical value, a normal shock wave will, in general, be expected (Figure 1b). As mentioned in the Introduction, this wave increases in strength as M_g is lowered, until it becomes stationary in the nozzle ... the "shock-in-nozzle" configuration (Figure 1c). Finally, for very low shock speeds, the nozzle flow becomes completely subsonic and no secondary waves can exist. Curves illustrating the regions of application of these various configurations are presented in Figure 2, for $A_4/A_1 = 1$.

The actual analyses are worked out in Appendix I. The ideal theory for the expansion wave configuration has been previously demonstrated for $A_4/A_1 > 1$ and $A^*/A_1 = 1$ (References 2 and 3)*, but is reviewed both as a background for the other configurations, and to show its logical extension to all values of A_4/A_1 and A^*/A_1 . It is seen that the solution of the basic shock tube relation is available in closed form for this case. Indeed, an "equivalent standard shock tube" may be defined as shown in Appendix I. A. In this analysis it is not necessary to consider the speed of the secondary expansion wave itself, for the ratio of the parameters across the wave depends only on the flow Mach numbers on either side. For the normal shock wave configuration, however, the expression for the pressure ratio across the secondary shock involves the speed of the shock wave, and elimination of this speed in terms of the flow Mach numbers involves the solution of a high degree algebraic equation. The resulting calculation becomes so unwieldy that the usefulness of a closed form solution for the normal shock case is questionable. The procedure presented provides a relatively fast means of calculating performance, avoiding the necessity for iteration by not attempting to solve the direct problem in which M_g is specified. The shock-in-nozzle configuration is an extension of the shock wave case, simplified because the shock is stationary, but complicated because of the additional steady expansion.

Illustrative performance curves are shown in Figures 3 and 4 for the high M_g regions. These curves are for varying A^*/A_1 only. Figures 5a and 5b allow determination of the performance of a Helium-Air shock tube of any geometry, but for the expansion wave case only. Figure 12 shows the low M_g solution for a Nitrogen-Air shock tube again

* Reference 3 also considers the energy aspects of the problem ... considerations of energy transfer efficiency across the various wave systems affords a useful qualitative understanding of the whole process.

with varying A^*/A_1 only. Since the effect of the nozzle depends solely on A_4/A_1 for the "subsonic nozzle" configuration, that part of the curves of Figure 12 corresponds to the conventional shock tube curve.

The dividing line between the regions of application of the expansion wave and shock wave model does not readily come out of the expansion wave theory. It is arrived at by taking the limit of the normal shock wave theory, that is, where the secondary shock has zero strength

$$(M_{3,1} - \frac{u_{s2}}{a_{3,1}} = 1) \quad .$$

The lower limit for the normal shock model is where the shock first becomes stationary ($u_{s2} = 0$); the lower limit for the shock in nozzle theory is where the shock is at the throat and has zero strength. These boundaries are depicted in Figure 2 for shock tubes with varying A^*/A_1 .

III. APPLICATION OF AREA CHANGE AT THE DIAPHRAGM

One of the first studies of area change at the diaphragm was that of Lukasiwicz⁴. Both he and subsequent authors^{2,3} have been primarily concerned with the interesting fact that area change may be used to increase available shock speed for a given shock tube. This effect is a maximum for $A^*/A_1 = 1$ and $A_4/A_1 \rightarrow \infty$; however, from Figures 5a and 5b it is seen that a small increase in A_4 produces very nearly the same effect as the limit of infinitely large A_4/A_1 . The actual per cent increase in M_s over that of a conventional shock tube is a complicated function of M_s itself, but Figures 5a and 5b illustrate it to be of the order of ten per cent for a practical shock tube operating at near maximum pressure ratio. Information contained in Reference 3 shows that this figure is reasonable for shock tubes operating with other gas combinations. While the figure of ten per cent is appreciable in some instances, other techniques will generally produce more spectacular increases (i. e., different gas combinations, a heated driver, or a double diaphragm shock tube⁵).

A study was made of the use of area change at the diaphragm for the economical operation of large shock tubes. The diaphragm cost (proportional to $A^{3/2}$) turns out to be negligible when compared with that of the driver gas at the high pressure ratios where cost becomes significant. The gas volume is proportional to $P_4 A_4$, and it is readily shown that A^*/A_1 must be unity for optimum economy at a given M_s . The optimum value of A_4/A_1 , however, is not obvious, and curves of A_4/A_1 versus $P_4 A_4$ must be plotted for each value of M_s . Such curves are presented in Figure 6. It is seen that the value of A_4/A_1 at which $P_4 A_4$ is a minimum is a function of M_s as expected. The trend to higher optimum values of A_4/A_1 with increasing M_s is a result of the asymptotic behavior of the basic performance curves, such that the high values of M_s cannot reasonably be achieved in a plain shock tube where the M_s asymptote is low. For operation at moderate M_s it is seen from Figure 6 that a conventional shock tube ($A_4/A_1 = 1$) is not far off optimum for economical operation. For operation at high M_s , however, the above logic shows that the use of $A_4/A_1 > 1$ is most efficient, and Figure 6 illustrates that factors of ten or greater decrease in operating cost can be realized for a Helium-Air shock tube. Studies of this nature aide in the choice of

driver size, and they may result in a significant decrease in operating cost for a large shock tube.

The use of area change can provide additional flexibility in shock tube operation. Since the nozzle used in the ideal theory has no characteristic axial length, it may be assumed to have negligible length and to consist merely of a drilled plate placed perpendicular to the flow at the diaphragm station. Separation losses and consequent violation of the assumptions inherent in the ideal theory might now be expected, however the experimental evidence presented in the next section shows these losses to be negligible. Thus the use of a simple insert in a conventional shock tube enables any reduced area diaphragm shock tube to be easily obtained. The insert plates provide an additional control on Reynolds number, and they are a logical way to produce low shock speeds.*

Figures 3, 4, and 12 show the performance of such a modified shock tube; Figure 7 shows the diaphragm insert section used for obtaining the experimental results presented in this report. Placement of the drilled plate just far enough downstream to clear the opening diaphragm provides model and instrument protection from possible diaphragm fragments, and circumvents the rippage and uncertain breaking pressure problems associated with the alternative use of very thin diaphragms. For large shock tubes it may be desirable to reduce the diaphragm size and thickness, and this may be facilitated by clamping the diaphragm directly over the plate orifice.

* In addition, the ideal theory for the normal shock wave configuration predicts a jump in pressure after the primary shock wave, and a jump back to essentially the initial pressure after the passage of the secondary wave. This pressure pulse might have useful application, but the experimental evidence indicates that it is not to be expected for small shock tubes operating at reasonable pressure levels.

IV. EXPERIMENTAL RESULTS

An experimental study of area change near the diaphragm of a shock tube was undertaken in the GALCIT 3 inch square tube. The purpose of this study was to check the validity of the ideal theory, both by observing the basic performance of a modified shock tube, and by an investigation of the secondary waves. This shock tube has an area ratio of $A_4/A_1 = 0.855$, and was modified by the diaphragm insert of Figure 7. Nitrogen-Air runs were made with various insert plates giving a range of A^*/A_1 from 0.855 to 0.03. Varying the initial pressure from 5 - 500 mm. gave enough range in M_s to enable measurements to be made in all of the flow regimes involving secondary waves. A description of the shock tube and instrumentation used is presented in Appendix II.

The ideal theory predicts initial pressure ratio as a function of M_s with no attenuation. In order to experimentally verify these curves with M_s measured some distance down the tube, the attenuation must be taken into account. Thus, for the initial measurements, M_s was obtained at two stations spaced a maximum distance apart along the shock tube.

The resulting values of per cent attenuation per foot showed considerable scatter, and only order of magnitude correlation with theory⁶. This was presumably due to the uncertain effect of diaphragm opening time, and the relatively short distance over which it was possible to make the measurements. The measurements did show, however, that for a given M_s there is no consistent or appreciable dependence of attenuation on A^*/A_1 . This means that a value of M_s , measured in a modified shock tube, may be corrected for attenuation by adding to it the difference between the ideal and the best fit experimental curves for the unmodified shock tube with the same M_s . Figure 8 presents the original P_4/P_1 versus M_s data. (Note that the ideal theory curves are plotted for $A_4/A_1 = 0.855$ and thus differ from those of Figure 12.) Figure 9 shows the data replotted with the small attenuation correction applied. The experimental points were obtained on different days and are accurate to the order of the symbol size. A few points, for which the diaphragms were only partially opened, fell understandably low and have been omitted.

The agreement between the theory and the corrected experimental results is seen to be excellent for the shock-in-nozzle configuration. The

experimental points also appear to fair into the ideal theory curve for the expansion wave case. The validity of the ideal theory for the expansion wave configuration was also noted by Alpher and White⁷, who made measurements at high M_g in a shock tube with various values of A_4/A_1 and observed that the change in basic performance agrees with the theory. These results emphasize the general validity of the ideal theory and they imply that any nozzle plate losses are insignificant (from the standpoint of the ideal theory). Indeed, nozzle losses could be easily added into the theory (if the experimental data were available), but it is felt that the remaining uncertainties in attenuation are of more significance.

The discrepancy between theory and experiment for the normal shock configuration remains unexplained. In order to shed more light on this situation the actual details of the secondary waves were investigated for the normal shock wave and expansion wave configurations. The primary tool for this investigation was the fine unheated wire⁸, for which the analysis techniques are discussed in Appendix II. Tungsten wires with diameters ranging from 1.5 - 0.2 mils. were used, affording a wide range of sensitivity. The wire response was expected to exhibit a sharp change in slope when a secondary shock wave passes over it, or a gradual change of slope through an expansion wave. The slope on each side of the secondary wave, together with the expected time of arrival of the wave at the gauge, was calculated and compared with the experimental results. The use of slope ratios eliminates some of the wire calibration properties, and it is felt that the resulting slope predictions are accurate to within ten per cent. Principle errors in this calculation are due to end losses, the cumbersome calculation procedure, and the extension of existing hot wire data to the current situation.

Typical results are presented in Figure 10, where the relevant theoretical and experimental slope ratios are also listed. Figure 10a, for the expansion wave regime, shows reasonable agreement, both in times and slope ratios. The normal shock wave results, however, did not show the expected change in slope. Indeed, there was no definite slope change observed for this configuration with any of the wires tested, and the predicted slope ratio was always considerably different from the observed value, indicating that the secondary normal shock model is incorrect.

An examination of the slope ratios for the normal shock configuration revealed that the experimental slope is always nearer to the predicted values of s_2/s_3 , than it is to s_2/s_3 . Since no change in slope occurs after the contact surface, this might indicate that the secondary shock occurred at the contact surface. To check this unlikely possibility, and to provide additional information concerning this configuration, static pressure measurements were made using a commercially available crystal transducer. (Kistler Piezo-Calibrator Model 2 with S. L. M. pickup) A sample trace is shown in Figure 11a. These traces did not show the expected sharp pressure drop through the secondary shock wave. Instead, they showed the gradual pressure rise that has been observed in conventional shock tubes.¹⁴

It is unlikely that the shock wave is masked out at the wall by boundary layer inter-action, but total pressure measurements were made to check this possibility. A small total pressure probe was constructed, using a Barium Titanite crystal modeled after that of Reference 9. A trace from this instrument is shown in Figure 11b. There is a suggestion of a drop in total pressure, but it is not conclusive in that it is unfortunately obscured by hash from the diaphragm and a natural frequency of the crystal mounting. Since the total pitot pressure is relatively insensitive to the existence of a secondary normal shock, this study was not pursued further than a few unsuccessful attempts to clean up the response. It is felt that the measurements tended to support the previous findings of no secondary normal shock wave.

A final experimental study was made with a four inch wooden nozzle attached to the downstream side of the 0.03 inch orifice plate. The purpose of this nozzle was to check the effect of the plate itself on the existence of the normal shock wave. Fine wire results were essentially identical with and without the nozzle on the plate. The points on the P_4/P_1 versus M_s curves of Figures 8 and 9 fall a little high, but the results are too few to be conclusive, and are probably due to a change in the attenuation history. A smoothly contoured nozzle appears to produce the same shock tube flow as the orifice plates.

V. FRICTIONAL THEORY FOR THE NON-ISENTROPIC SECONDARY WAVE CONFIGURATIONS

The experimental results of the last section indicate that the normal shock model is marginal for predicting M_g and thus conditions in region 2, and quite inadequate for describing the flow details upstream of the contact surface. To agree with the measurements, a new theory for this non-isentropic configuration is needed which will predict no static pressure jump and no sharp slope changes in the fine wire response. The new theory must, in addition, predict nearly the same basic performance as the ideal theory, and must fair logically into the valid solutions for the other configurations.

The starting point for the new theory might logically be to consider the secondary shock to be replaced by many weak shocks, resulting in a spread out compression region. The existence of a spread out compression region in a supersonic duct has been studied in connection with wind tunnel nozzles^{10, 11}. Indeed, in Reference 11 it is pointed out that the shock wave reflecting off the end of a shock tube has been observed to take this form. It would seem quite possible that the interaction between the secondary shock wave and the thick boundary layer from the diaphragm could also result in a spread out region for the configuration under study.

Appendix III. A. shows the relevant calculations for a new model based on the replacement of the secondary shock wave with such a stationary interaction region. The functional relationship of the flow variables across this region is that of a Fanno curve, and the model will be referred to as the Fanno process model. The relationship is obtained by writing the energy and continuity equations for the average cross-sectional flow values, replacing the momentum equation by a statement of the overall pressure ratio. This allows for arbitrary frictional processes and still enables the shock tube performance to be calculated, although the details in the Fanno region itself remain arbitrary. It should be mentioned that in the calculations that lead to the Fanno model, the exit Mach numbers may come out to be subsonic. Since a frictional process in a constant area duct cannot result in a transition from supersonic to subsonic flow¹², a stationary shock wave must exist within the Fanno regions for these cases.

Figure 12 shows the results of the Fanno process model for $A_4/A_1 = 1$. * Figure 9 shows the remarkable agreement between this model and the experimental data. It is seen that the new curves fair in with both the expansion wave and subsonic nozzle solutions, even though they may cross the ideal theory curve before doing so. It is interesting to note that there is no slope discontinuity corresponding to that which occurs at the junction of the normal shock and shock-in-nozzle solutions, and that the shock-in-nozzle solution itself is unchanged by the new theory.

An attempt was made to choose a combination of shock tube gases so as to further separate the two solutions and thus experimentally check the validity of the Fanno model under more severe conditions, but the relative M_s difference was quite insensitive to the gases used. The insensitivity of the basic performance to the model used is further exemplified by the fact that solutions which consist of part stationary Fanno process and part drifting secondary normal shock lie in the small enclosed region between the two pure solutions.

The agreement with the detailed flow measurements is also improved with the Fanno model. The calculated fine wire slope ratio for Figure 10b is -5.4 as compared with an experimental value of -7.7, and an ideal theory value of -2.0. The static pressure is now expected to remain constant, and while it actually tends to rise, this feature has been observed in plain shock tubes¹⁴ and is thus not directly connected with the area change.

There is another interesting reason for accepting the Fanno model. From Reference 13 it can be inferred that the nozzle starting process is important for times as short as those involved in shock tube operation, and, in fact, the nozzle may not attain fully developed supersonic flow before the end of the run. A quasi-steady model of this situation might be to consider isentropic flow to a given station in the nozzle exit, and a Fanno process through the remainder of the nozzle. As time proceeds, more and more of the nozzle will be occupied with the isentropic flow and the Fanno region will move out of the nozzle. It is shown in Appendix III. B that for this model the flow parameters downstream of the Fanno process are independent of the location of the beginning of the Fanno process in the nozzle, and depend only on the overall pressure ratio across the whole

* The plain shock tube curve was obtained with the use of a Fanno process from sonic velocity, replacing that part of the driver expansion wave which was downstream of the diaphragm.

region. Thus this quasi-stationary model must give results identical with the previous Fanno curve model, a fact which lends further credence to that model.

The simple stationary Fanno model has shown appreciably better agreement with experiment than the ideal theory model, indeed the remaining unexplained differences are of the same nature as those which have been observed in a plain shock tube¹⁴. The use of the basic Fanno process idea, inter-related with a time-dependent mechanism for attenuation, may be a key to more complete understanding of these flows. The Fanno region may be combined with non-steady waves, may be allowed to drift down the tube, or may be assumed to grow with time (i. e., the downstream boundary moving with the speed of the contact surface). In the latter case, the time dependent equations may be simplified by the introduction of a conical flow parameter, but the detailed history of one variable is needed to replace the momentum equation, and solution difficulties arise for all but the most simple assumed profiles.

A model, consisting of a stationary Fanno curve with a weak expansion wave downstream, was tried in an attempt to explain the wall pressure rise with time that existed for all shock tube configurations. Some agreement with experiment was obtained, but more detailed measurements are necessary in order to justify the existence of this type of model, and to provide a thorough understanding of the Fanno-type of frictional process in a shock tube. In particular, it would be useful to learn more of when this process may be advantageously used in place of the more complicated boundary layer approaches.

VI. CONCLUSIONS

1. The ideal theory presented in this report accurately predicts M_s for shock tubes with area change near the diaphragm, with the exception of those configurations where a moving secondary shock wave is expected.

2. In cases involving moving secondary shock waves, the stationary Fanno curve model discussed in this report accurately predicts M_s and provides agreement with flow details to the same degree that flow measurements agree with theory for a plain shock tube. Further refinement of the theory, to coincide more closely with observation, awaits a more detailed experimental investigation. It is suggested that the Fanno-type of frictional process might have useful application to other shock tube problems.

3. It appears that the most practical applications of area change at the diaphragm are the use of A_4/A_1 for increased M_s , and the use of simple insert plates to provide an additional control on shock tube performance. In addition to these, and certain advantages that may occur in large shock tubes, area change near the diaphragm offers an interesting research tool for a further understanding of shock tube flows.

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17. Rabinowicz, J.: "Aerodynamic Studies in the Shock Tube". GALCIT Hypersonic Research Project, Memorandum No. 38, June 10, 1957.
18. Laufer, J.; R. McClellan: "Measurements of Heat Transfer from Fine Wire in Supersonic Flows". J. of Fluid Mech., Vol. 1, Part 3, p. 276, Sept., 1956.
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APPENDIX I

IDEAL THEORY CALCULATION PROCEDURES

It may readily be shown from one-dimensional shock theory that

$$P_a/P_b = 1 + (2\gamma/\gamma + 1) (M_b^2 - 1) \quad , \quad (1)$$

transforming velocity so as to apply to a shock tube results in

$$P_2/P_1 = 1 + (2\gamma/\gamma + 1) (M_s^2 - 1) \quad . \quad (2)$$

A similar procedure results in

$$u_2/a_1 = (2/\gamma_1 + 1) (M_s - \frac{1}{M_s}) \quad . \quad (3)$$

Hence,

$$u_2/a_1 = \left(\frac{P_2}{P_1} - 1 \right) \left[\gamma_1 \frac{(\gamma_1 + 1)}{2} \left(\frac{P_2}{P_1} - 1 \right) + \gamma_1^2 \right]^{-\frac{1}{2}} \quad . \quad (4)$$

The Riemann invariant applied across a non-steady isentropic wave results in:

$$P_a/P_b = \left[\rho_a/\rho_b \right]^\gamma = \left[a_a/a_b \right]^{2\gamma/\gamma-1} = \left[\frac{1 + \frac{\gamma-1}{2} M_b^2}{1 + \frac{\gamma-1}{2} M_a^2} \right]^{2\gamma/\gamma-1} \quad , \quad (5)$$

while the energy relation applied across a steady isentropic process results in

$$P_a/P_b = \left[\rho_a/\rho_b \right]^\gamma = \left[a_a/a_b \right]^{2\gamma/\gamma-1} = \left[\frac{1 + \frac{\gamma-1}{2} M_b^2}{1 + \frac{\gamma-1}{2} M_a^2} \right]^{\gamma/\gamma-1} \quad . \quad (6)$$

In the following analyses, extensive use is made of the tables for the parameter relationships across a stationary normal shock wave, and tables for steady isentropic flow with area change^{15, 16}. The calculations are arranged so that only these tables and curves of Eq. (4) need be obtained in order to calculate the complete performance of a given shock tube. In the following calculations, perfect gases have been assumed throughout.

A. Expansion Wave Configuration

Referring to Figure 1a, the overall pressure ratio may be written as

$$P_4/P_1 = (P_4/P_{4'}) (P_{4'}/P_{3'}) (P_{3'}/P_3) (P_3/P_2) (P_2/P_1) \quad (7)$$

Applying Eq. (5) across the non-steady expansion waves (regions 4-4' and 3'-3), Eq. (6) through the supersonic nozzle (4'-3'), and applying the pressure boundary condition, $P_3 = P_2$, yields:

$$(P_4/P_1) = \left[1 + \frac{\gamma_4 - 1}{2} M_{4'}^2 \right]^{(2\gamma_4)/(\gamma_4 - 1)} \left[\frac{1 + \frac{\gamma_4 - 1}{2} M_{3'}^2}{1 + \frac{\gamma_4 - 1}{2} M_{4'}^2} \right]^{(\gamma_4)/(\gamma_4 - 1)} \\ \times \left[\frac{1 + \frac{\gamma_4 - 1}{2} M_3^2}{1 + \frac{\gamma_4 - 1}{2} M_{3'}^2} \right]^{(2\gamma_4)/(\gamma_4 - 1)} (P_2/P_1) \quad (8)$$

Now, extending the results of Reference 2, define an "amplification factor":

$$g = \left[\frac{1 + \frac{\gamma_4 - 1}{2} M_{4'}^2}{1 + \frac{\gamma_4 - 1}{2} M_{3'}^2} \right]^{(\gamma_4)/(\gamma_4 - 1)} \left[\frac{1 + \frac{\gamma_4 - 1}{2} M_{3'}^2}{1 + \frac{\gamma_4 - 1}{2} M_{4'}^2} \right]^{(2\gamma_4)/(\gamma_4 - 1)} \quad (9)$$

g is a constant, depending only on the driver gas and the parameters A_4/A_1 and A^*/A_1 . It may readily be determined with the use of tables^{15, 16}. Curves of g for all values of the area parameters are shown in Figure 5a for a Helium-Air shock tube. Since A_4/A_1 cannot be less than A^*/A_1 by definition, the line where these ratios are equal forms the left-hand boundary and each curve for a given value of A_4/A_1 must terminate on this boundary. Previous work^{2, 3} has been concerned only with the case where $A^*/A_1 = 1$; inclusion of the effect of A^*/A_1 results in a complete surface as shown.

Equations (8) and (9) result in

$$P_4/P_1 = \frac{1}{g \left(\frac{A_4}{A_1}, \frac{A^*}{A_1}, \gamma_4 \right)} \left[1 + \frac{\gamma_4 - 1}{2} M_3 \right]^{(2\gamma_4)/(\gamma_4 - 1)} (P_2/P_1) \quad (10)$$

The velocity boundary condition, $u_2 = u_3$, enters through M_3 :

$$M_3 = (u_3/a_3) = (u_2/a_3) = (u_2/a_1)(a_1/a_4)(a_4/a_4)(a_4/a_3)(a_3/a_3) \quad .$$

Using Eqs. (5), (6), and (9):

$$M_3 = (u_2/a_1)(a_1/a_4) g^{\frac{\gamma_4 - 1}{2\gamma_4}} \left(1 + \frac{\gamma_4 - 1}{2} M_3 \right) \quad (12)$$

Solving Eq. (12) for M_3 and inserting into Eq. (10), there is obtained

$$P_4/P_1 = (1/g)(P_2/P_1) \left[1 - \frac{\gamma_4 - 1}{2} (u_2/a_1) \left(\frac{a_1}{a_4} g^{-\frac{\gamma_4 - 1}{2\gamma_4}} \right) \right]^{-(2\gamma_4)/(\gamma_4 - 1)} \quad (13)$$

Thus, assuming a given M_s , one may calculate u_2/a_1 and P_2/P_1 from relations 2 and 3. Knowing g , P_4/P_1 may be calculated, and in a similar manner any of the flow parameters arrived at.

Reference should be made to the well known connection between the above theory and that of a standard shock tube. The standard shock tube equation may be obtained by putting $g = 1$ in Eq. (13):

$$P_4/P_1 = P_2/P_1 \left[1 - \frac{\gamma_4 - 1}{2} (u_2/a_1)(a_1/a_4) \right]^{-(2\gamma_4)/(\gamma_4 - 1)} \quad (14)$$

Comparing Eqs. (13) and (14), it is seen that the area-ratio tube produces the same shock speed for a given P_4/P_1 as a standard tube with the following modifications:

$$(a_1/a_4)_{\text{standard}} = g^{-\frac{\gamma_4 - 1}{2\gamma_4}} (a_1/a_4)_{\text{area ratio}} \quad (15)$$

$$(P_4/P_1)_{\text{standard}} = g (P_4/P_1)_{\text{area ratio}}$$

B. Normal Shock Wave Configuration

Starting as in the previous case, applying the pressure boundary condition and relations 5, 6, and 9, the overall pressure ratio becomes:

$$P_4/P_1 = (P_{3'}/P_3)(P_2/P_1) \frac{\left[1 + \frac{\gamma_4 - 1}{2} M_{3'}^2\right]^{(2\gamma_4)/(\gamma_4 - 1)}}{g}$$

$$= C_1 \left(\frac{A_4}{A_1}, \frac{A^*}{A_1}, \gamma_4 \right) (P_{3'}/P_3)(P_2/P_1) \quad (16)$$

where the notation is that of Figure 1b.

The velocity boundary condition results in

$$u_2/a_1 = u_3/a_1 = (u_3/u_{3'})(u_{3'}/a_{3'})(a_{3'}/a_{4'})(a_{4'}/a_1) \quad (17)$$

Again using Eqs. (5), (6), and (9):

$$u_2/a_1 = u_3/u_{3'} \left(\frac{a_4}{a_1} \frac{M_{3'}}{1 + \frac{\gamma_4 - 1}{2} M_{3'}^2} \right)^{(\gamma_4 - 1)/(2\gamma_4)}$$

$$= C_2 \left(\frac{A_4}{A_1}, \frac{A^*}{A_1}, \gamma_4, \frac{a_4}{a_1} \right) (u_3/u_{3'}) \quad (18)$$

Now consider region 3 - 3' and define the secondary shock velocity as u_{s2} . Transforming to a stationary shock system, the flow Mach number into the shock becomes

$$(M_{3'} - \frac{u_{s2}}{a_{3'}}) \quad .$$

Eq. (1) then yields:

$$P_3/P_{3'} = 1 + \frac{2\gamma_4}{\gamma_4 + 1} \left(\left[M_{3'} - \frac{u_{s2}}{a_{3'}} \right]^2 - 1 \right) \quad (19)$$

The calculation procedure is as follows:

(1) Assuming supersonic flow out of the nozzle, obtain

$M_{3'} \left(\frac{A_4}{A_1}, \frac{A^*}{A_1}, \gamma_4 \right)$ from the tables and evaluate C_1 and C_2 in Eqs. (16) and (18).

(2) Now assume a value for u_{s2}/a_{31} and calculate P_3/P_3 from Eq. (19). Knowing the flow Mach number into the shock $M_{31} = (u_{s2}/a_{31})$, calculate or obtain from the tables the exit flow Mach number

$M_3 = (u_{s2}/a_3)$, and the speed of sound ratio a_{31}/a_3 .

(3) M_3 may be found from the relation:

$$M_3 = \text{exit flow Mach number} + \frac{u_{s2}}{a_{31}} (a_{31}/a_3) \quad (20)$$

The velocity ratio across the moving shock becomes:

$$u_3/u_{31} = M_3/M_{31} (a_3/a_{31}) \quad (21)$$

(4) Knowing u_3/u_{31} , Eq. (18) determines u_2/a_1 , Eqs. (4) and (2) determine P_2/P_1 and M_s . It is suggested that the P_2/P_1 versus u_2/a_1 curves referred to at the beginning of the section be used to determine P_2/P_1 , and M_s be either calculated from this value (Eq. 2) or picked out of the tables.

(5) With P_2/P_1 and P_3/P_3 known, P_4/P_1 is readily calculated from Eq. (16).

C. Shock-In-Nozzle Configuration

The velocity of the secondary shock was initially assumed for the shock wave configuration analysis. In this case, however, the shock is stationary, and it is necessary to assume the shock position in the nozzle. Denoting the nozzle area at this position as A_s , the equation analogous to Eq. (16) is

$$P_4/P_1 = \left[(P_{3''}/P_{31})(P_{31}/P_3) \right] P_2/P_1 \frac{\left[1 + \frac{\gamma_4 - 1}{2} M_{31}^2 \right]^{(2\gamma_4)/(\gamma_4 - 1)}}{g_s} \quad (22)$$

where $g_s = g \left(\frac{A_4}{A_s}, \frac{A^*}{A_s} \right)$ and the notation is in Figure 1c.

The velocity boundary condition yields:

$$u_2/a_1 = u_3/a_1 = M_3 (a_3/a_{31})(a_{31}/a_{3''})(a_{3''}/a_{41})(a_{41}/a_4)(a_4/a_1) \quad (23)$$

Applying Eqs. (5), (6), and (9):

$$u_2/a_1 = \left[M_3 (a_3/a_{3'}) (a_{3'}/a_{3''}) \right] (a_4/a_1) \frac{g_s (\gamma_4 - 1)/(2\gamma_4)}{1 + \frac{\gamma_4 - 1}{2} M_{3''}} \quad (24)$$

The knowledge of A_4/A_s , A^*/A_s , and γ_4 enables g_s and $M_{3''}$ to be calculated^{15, 16}. The shock tables and the value of $M_{3''}$ then give $P_{3''}/P_{3'}$, $a_{3'}/a_{3''}$, and $M_{3'}$. Allowing the subsonic flow at station 3' to expand, M_3 , $P_{3'}/P_3$, and $a_3/a_{3'}$, can be obtained by using the tables and the value of $M_{3'}$ and A_s/A_1 . Thus u_2/a_1 can be found from Eq. (24), and P_2/P_1 then obtained from the curves of Eq. (4). The value of P_2/P_1 and the shock tables yield M_s , and P_4/P_1 is determined from Eq. (22).

D. Subsonic Nozzle Configuration

In this case the nozzle flow is completely subsonic and conditions between regions 4' and 3 depend only on A_4/A_1 . The performance may be calculated by the procedure of Appendix I. A, putting conditions in region 3' equal to those in region 3 (since no secondary expansion wave exists), and choosing the subsonic value for M_3 . As expected, the resulting relations are identical with those for a plain shock tube when $A_4/A_1 = 1$, and there is an effect on performance only if $A_4/A_1 \neq 1$.

APPENDIX II

EXPERIMENTAL EQUIPMENT AND INSTRUMENTATION

The GALCIT three inch square shock tube was used for the experimental measurements. This is essentially the shock tube reported in Reference 17, but the pressure seals have been improved and the two-dimensional nozzle at the end of the tube has been replaced by a dump chamber. The shock tube has a circular driver such that $A_4/A_1 = 0.885$, and was modified for this series of tests by the diaphragm insert section of Figure 7. 0.006 inch aluminum diaphragms were used primarily. They were scored to break under a driver pressure from 20 - 100 psi. It was found that careful scoring resulted in the ability to repeat a given M_s to within 10 per cent. Some copper diaphragms were used, but it was found that the measured attenuation was not affected, and the basic performance virtually unchanged.

The shock speed was detected with two thin film platinum gauges spaced two feet apart, their average distance from the diaphragm being 16.7 feet. The gauges had high resistance (500 Ω) in order to maximize sensitivity for the low values of M_s . The output was fed through low noise level amplifiers to a Berkeley counter in the standard fashion. 10^{-2} A. current provided enough sensitivity to readily obtain shock speeds as low as $M_s = 1.04$. For the attenuation measurements, two pairs of gauges were used, the center of the second pair four feet upstream of the center of the first. Each pair was connected in series, fed through an amplifier, differentiated, and fed into the period function of a Berkeley counter.

The initial pressure was measured with a Wallace and Tiernan 0 - 50 mm gauge and an aircraft type manometer. The driver pressure was measured with a 0 - 100 psi. bourdon gauge. All gauges were carefully calibrated with either a McCleod gauge or known reference volumes, and the shock tube leak rate was maintained at a relatively low level.

The fine wires were constructed as in Reference 8. A constant excitation current was applied and the voltage across the wire was

directly sensed by an oscilloscope. The wire acts like a calorimeter, and thus its response to a given flow change is roughly exponential with time. Thus, for small times relative to the characteristic time, the slope of the wire response will be constant; for very long times, the wire response will be a step function. It was found that a wire chosen so as to have the former response characteristic was the most practical for these tests.

Given a specific M_g and P_1 , all the flow variables could be calculated from the ideal theory. Approximate Nusselt numbers for the various flow regions could then be found from the calculated Reynolds and Mach numbers through the use of References 18 and 19. The Nusselt number is defined as

$$Nu = \frac{q d}{K (T_r - T_w)} \quad (1)$$

where d is the wire diameter, K the thermal conductivity of the fluid (evaluated at total temperature²⁰), T_r the recovery temperature, and T_w the initial wire temperature. The heat transfer per unit area of wire, q , may be shown to be directly proportional to s , the response slope⁸. Thus,

$$s_2/s_3 = q_2/q_3 = \frac{K_2 (T_{r_2} - T_1)}{K_3 (T_{r_3} - [T_{r_1} + \Delta T_2])} (Nu_2/Nu_3) \quad (2)$$

It may also be shown that⁸

$$\Delta T_2 = \frac{\Delta E_2}{a_1 I_w R_w} \quad (3)$$

where I_w is the wire current ($\approx 2(10)^{-3}$ A.), R_w the wire resistance at T_1 ($\approx 2\Omega$), a the coefficient of resistivity (taken as $0.004\Omega/^{\circ}\text{C}$ for tungsten), and ΔE the voltage jump on the experimental response trace. ΔT_2 was of the order of 50°C for the $1\frac{1}{2}$ mil. wire.

Eq. (2) predicts the slope ratio between regions 2 and 3 when the recovery factors and Nusselt numbers have been found. The equation may easily be extended to other regions if care is taken in obtaining the wire temperature at the beginning of each change in slope.

APPENDIX III

THE FANNO PROCESS MODEL

A. Constant Area Fanno Model

In this study, the secondary normal shock of Figure 1b is replaced with a stationary compression region supported by shock boundary layer interaction. This interaction region is assumed to take place in the constant area duct, and, as in the ideal theory, the entering flow (region 3') is assumed known.

Assuming a perfect gas, the steady energy equation is

$$(a_a/a_b)^2 = \frac{1 + \frac{\gamma-1}{2} M_b^2}{1 + \frac{\gamma-1}{2} M_a^2} = (P_a/P_b)(\rho_b/\rho_a) \quad (1)$$

The continuity equation, applied between regions 3 and 3' may be written as:

$$(P_3 M_3)/a_3 = (P_{3'} M_{3'})/a_{3'} \quad (2)$$

where it is understood that flow parameters describing the Fanno region are the average values across the cross section.

Applying Eq. (1) across the Fanno region, and substituting $a_3/a_{3'}$ from Eq. (2) yields

$$\left(1 + \frac{\gamma-1}{2} M_{3'}^2\right) \left[\frac{P_{3'} M_{3'}}{P_3}\right]^2 = M_3^2 \left(1 + \frac{\gamma-1}{2} M_3^2\right) \quad (3)$$

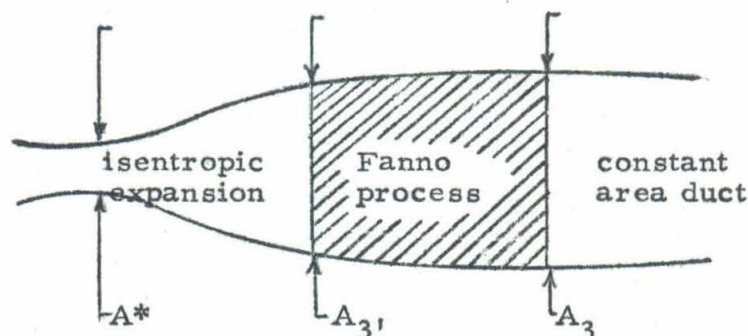
Solving for M_3

$$(\gamma-1) M_3^2 = -1 + \left[1 + 2(\gamma-1)\left(1 + \frac{\gamma-1}{2} M_{3'}^2\right) \left\{(P_{3'}/P_3) M_{3'}\right\}^2\right]^{\frac{1}{2}} \quad (4)$$

The shock tube performance is readily calculated by first assuming $P_{3'}/P_3$. Then, since $M_{3'}$ is known, M_3 may be found. From this information $a_3/a_{3'}$ is obtained from Eq. (2). u_3/a_1 may now be determined, and the performance calculated in a manner analogous to that of Appendix I.

B. Varying Area Fanno Model

For this model, the Fanno process is assumed to take place in the nozzle itself. The flow is assumed to expand isentropically from the sonic throat to some station 3' and then proceed along a Fanno curve to a station downstream of the nozzle. (See sketch.) The entering flow is known, as well as A^*/A_3 , $A_{3'}/A_3$, and P_*/P_3 .



The isentropic relation between the throat and station 3' gives

$$P_*/P_{3'} = \left[\rho_*/\rho_{3'} \right]^\gamma = \left[a_*/a_{3'} \right]^{2\gamma/\gamma-1} \quad (5)$$

Appropriately applying Eqs. (1) and (5), the overall density ratio may be written:

$$\rho_*/\rho_3 = (\rho_*/\rho_{3'}) (\rho_{3'}/\rho_3) = \left[\frac{1 + \frac{\gamma-1}{2} M_{3'}^2}{(\gamma+1)/2} \right]^{1/\gamma-1} \left[\frac{1 + \frac{\gamma-1}{2} M_{3'}^2}{1 + \frac{\gamma-1}{2} M_3^2} \right] \frac{P_{3'}}{P_3}; \quad (6)$$

but

$$P_{3'}/P_3 = (P_{3'}/P_*) (P_*/P_3) = \left[\frac{(\gamma+1)/2}{1 + \frac{\gamma-1}{2} M_{3'}^2} \right]^{\gamma/\gamma-1} (P_*/P_3) \quad (7)$$

Thus, collecting terms:

$$\rho_*/\rho_3 = \frac{(\gamma+1)/2}{1 + \frac{\gamma-1}{2} M_3^2} (P_*/P_3) \quad (8)$$

and the continuity equation and perfect gas relation yield:

$$M_3 = (\rho_*/\rho_3)(a_*/a_3)(A_*/A_3) = \left[\rho_*/\rho_3 \right]^{\frac{1}{2}} \left[P_*/P_3 \right]^{\frac{1}{2}} (A_*/A_3) . \quad (9)$$

Applying Eq. (8) to Eq. (9) and solving the algebraic relation:

$$(\gamma - 1) M_3^2 = -1 + \left[1 + (\gamma^2 - 1) \left[(P_*/P_3)(A_*/A_3) \right]^2 \right]^{\frac{1}{2}} ; \quad (10)$$

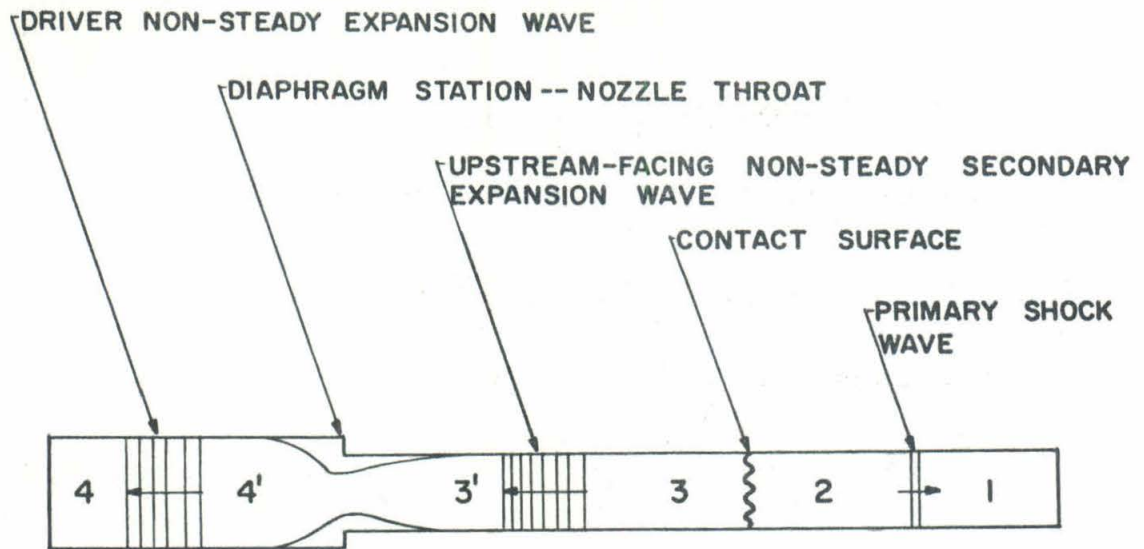
but, again, applying continuity and Eq. (8):

$$u_3/a_* = (\rho_*/\rho_3)(A_*/A_3) = \frac{(\gamma+1)/2}{1 + \frac{\gamma-1}{2} M_3^2} (P_*/P_3)(A_*/A_3) , \quad (11)$$

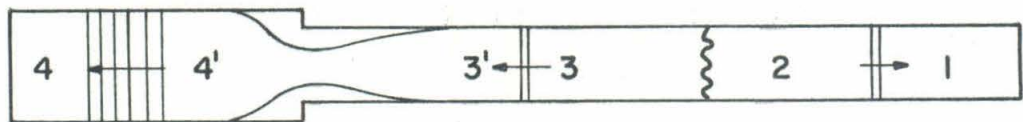
and Eq. (11) together with Eq. (10)

$$u_3/a_* = \frac{(\gamma+1) \left[(P_*/P_3)(A_*/A_3) \right]}{1 + \left[1 + (\gamma^2 - 1) \left[(P_*/P_3)(A_*/A_3) \right]^2 \right]^{\frac{1}{2}}} . \quad (12)$$

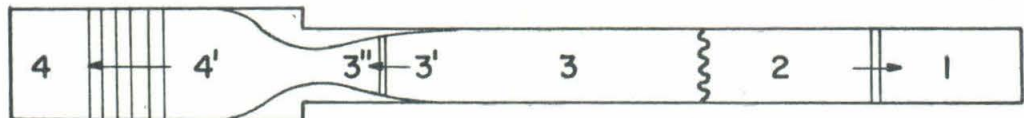
Eqs. (8), (10), and (12) demonstrate that the exit flow is independent of the position of station 3'. Thus the Fanno process could be completely outside of the nozzle, so that this solution is identical to that of part A above.



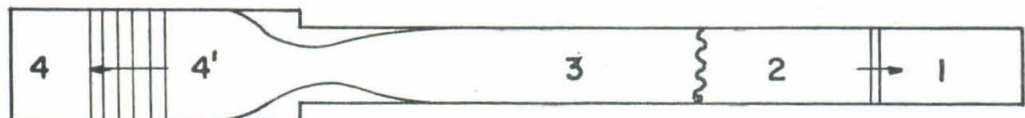
a. EXPANSION WAVE CONFIGURATION



b. SHOCK WAVE CONFIGURATION



c. SHOCK-IN-NOZZLE CONFIGURATION



d. SUBSONIC NOZZLE CONFIGURATION

FIG. 1 SCHEMATIC SHOCK TUBE CONFIGURATIONS

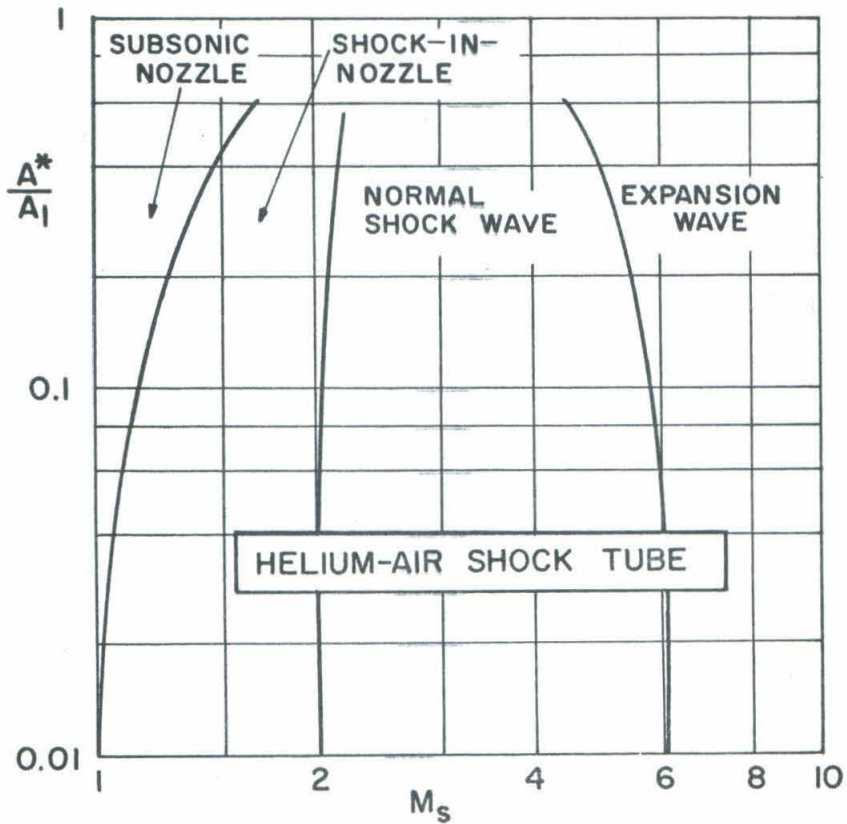
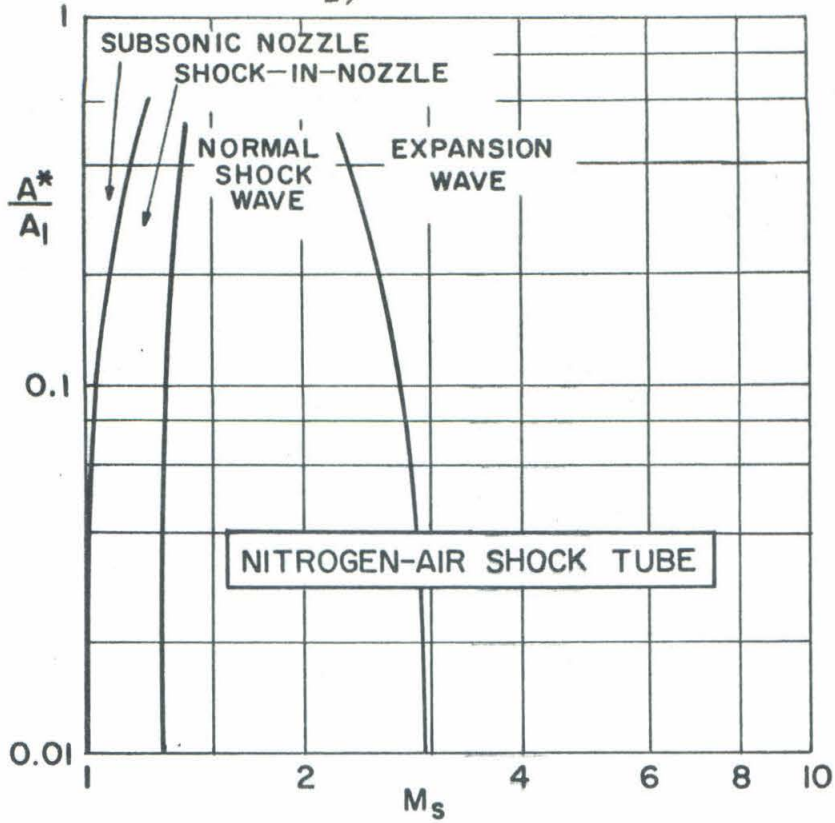


FIG. 2 CONFIGURATION BOUNDARIES FOR $A_4/A_1 = 1$

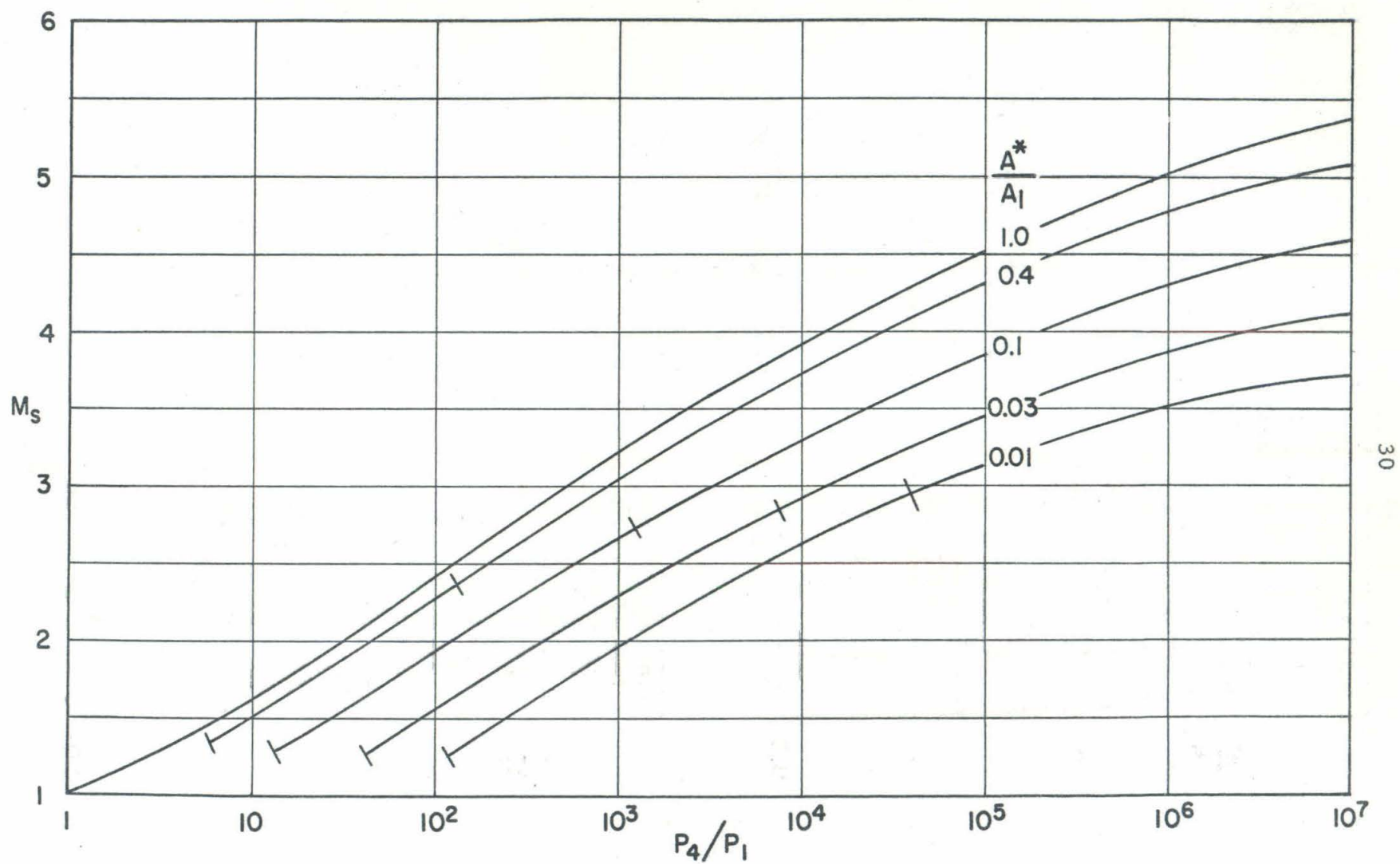


FIG. 3 BASIC PERFORMANCE--NITROGEN-AIR SHOCK TUBE; $A_4/A_1 = 1$

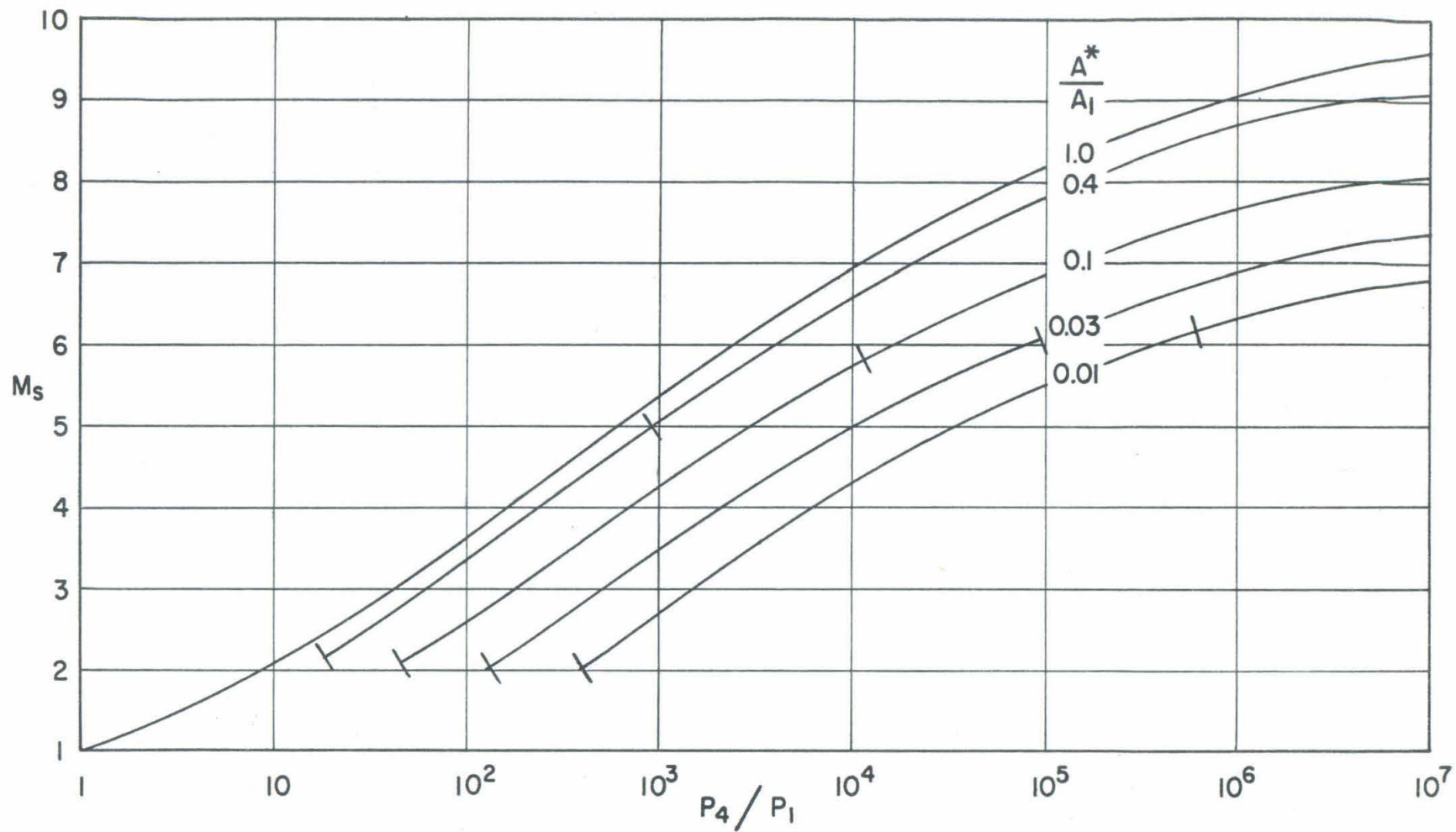


FIG. 4 BASIC PERFORMANCE -- HELIUM-AIR SHOCK TUBE; $A_4/A_1 = 1$

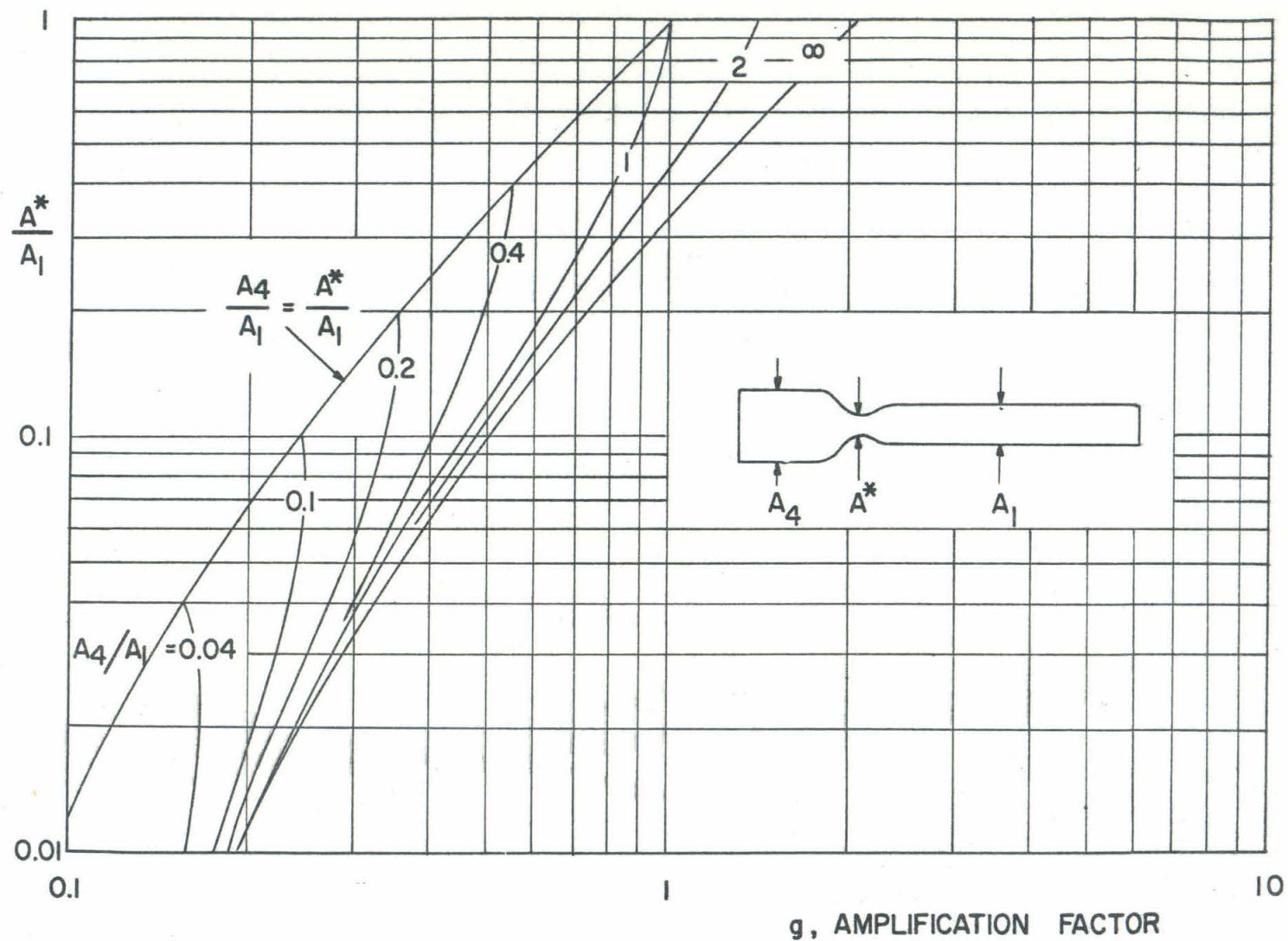


FIG. 5(a) AMPLIFICATION FACTOR FOR $\gamma = 5/3$

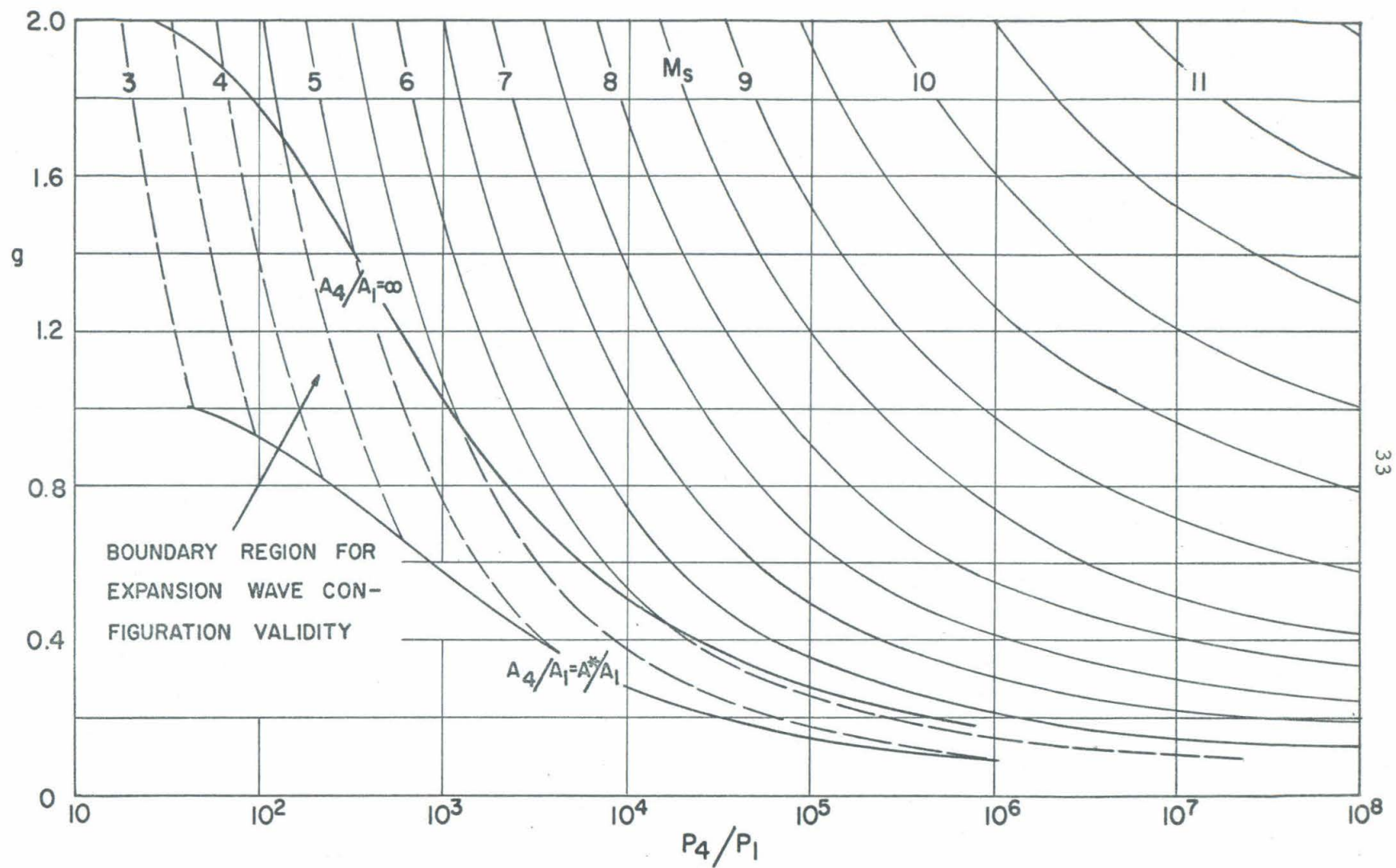


FIG. 5(b) BASIC PERFORMANCE -- HELIUM-AIR SHOCK TUBE; EXPANSION WAVE CONFIGURATION

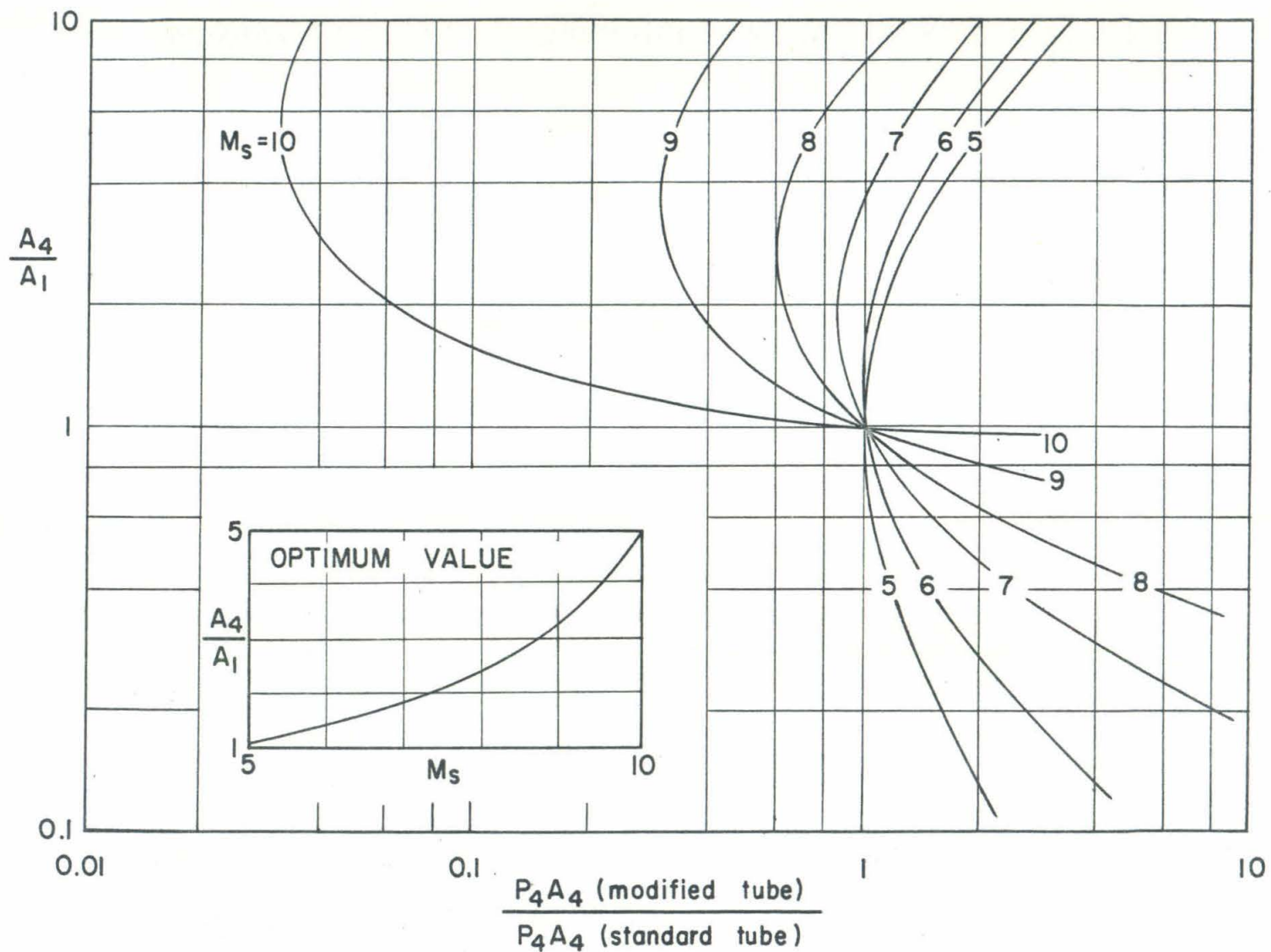


FIG. 6 OPTIMIZATION OF A_4/A_1 FOR ECONOMICAL OPERATION -- HELIUM-AIR SHOCK TUBE

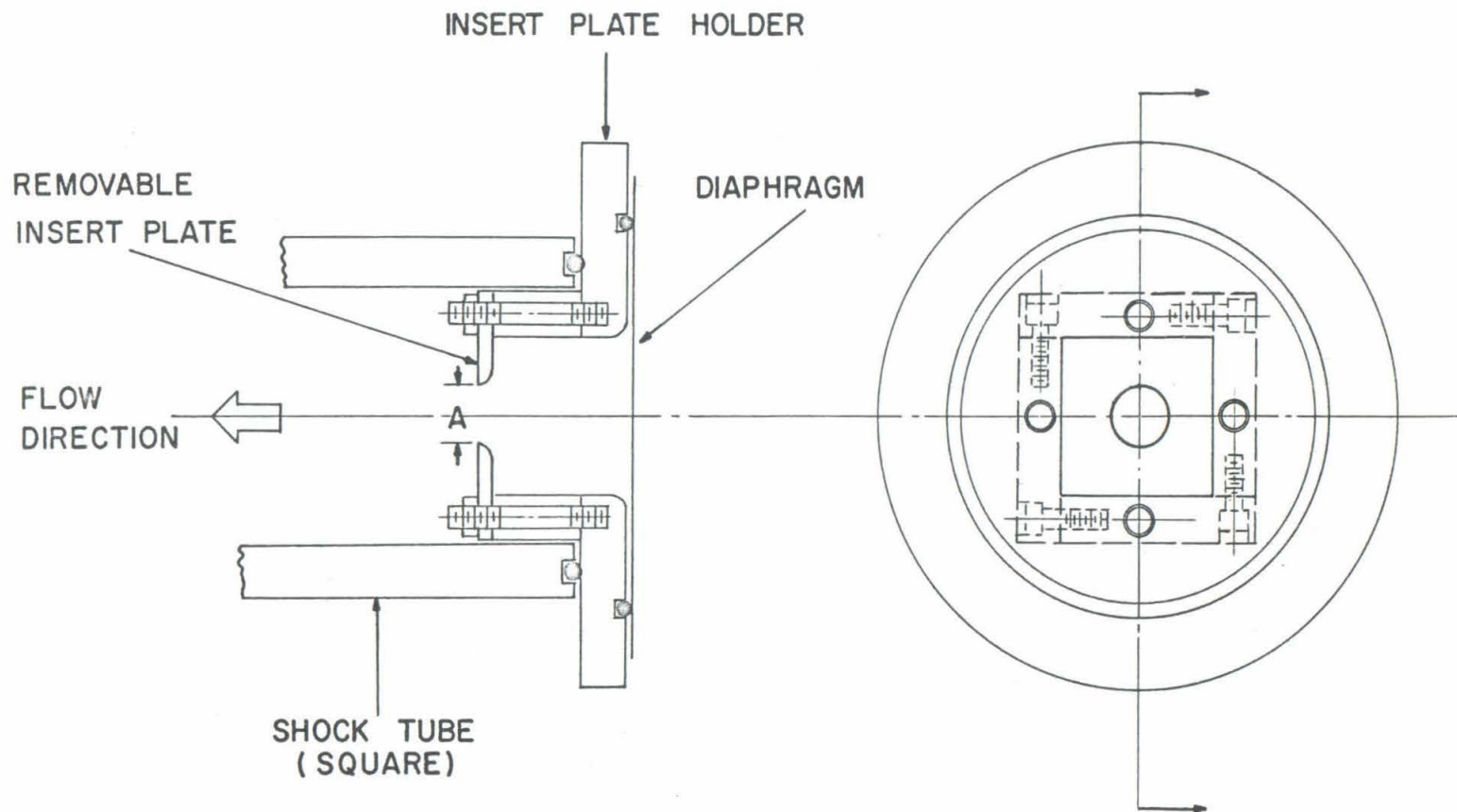


FIG. 7 DIAPHRAGM INSERT SECTION -- 1/2 SCALE

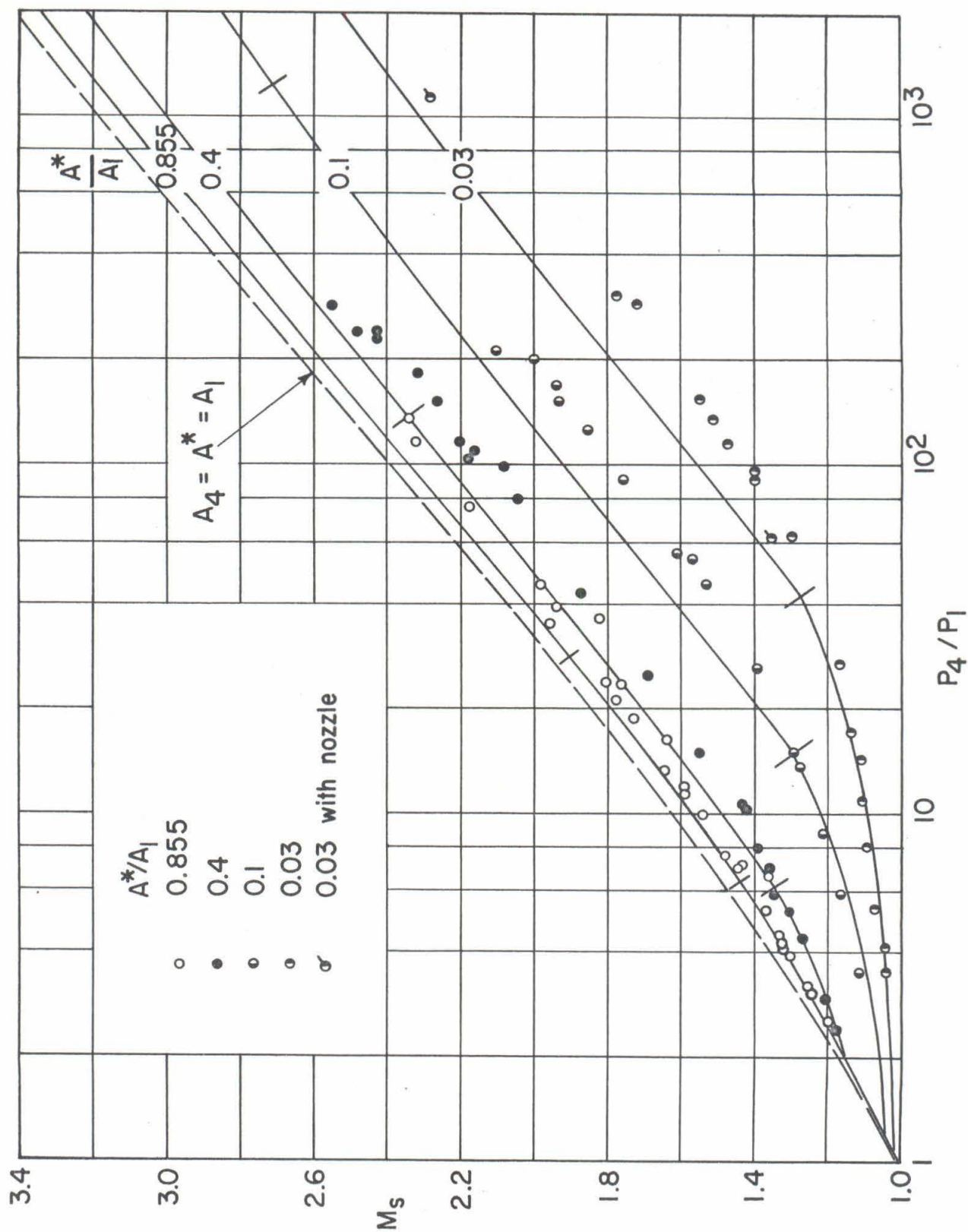


FIG. 8 UNCORRECTED DATA -- NITROGEN-AIR SHOCK TUBE;
 $A_4/A_1 = 0.855$

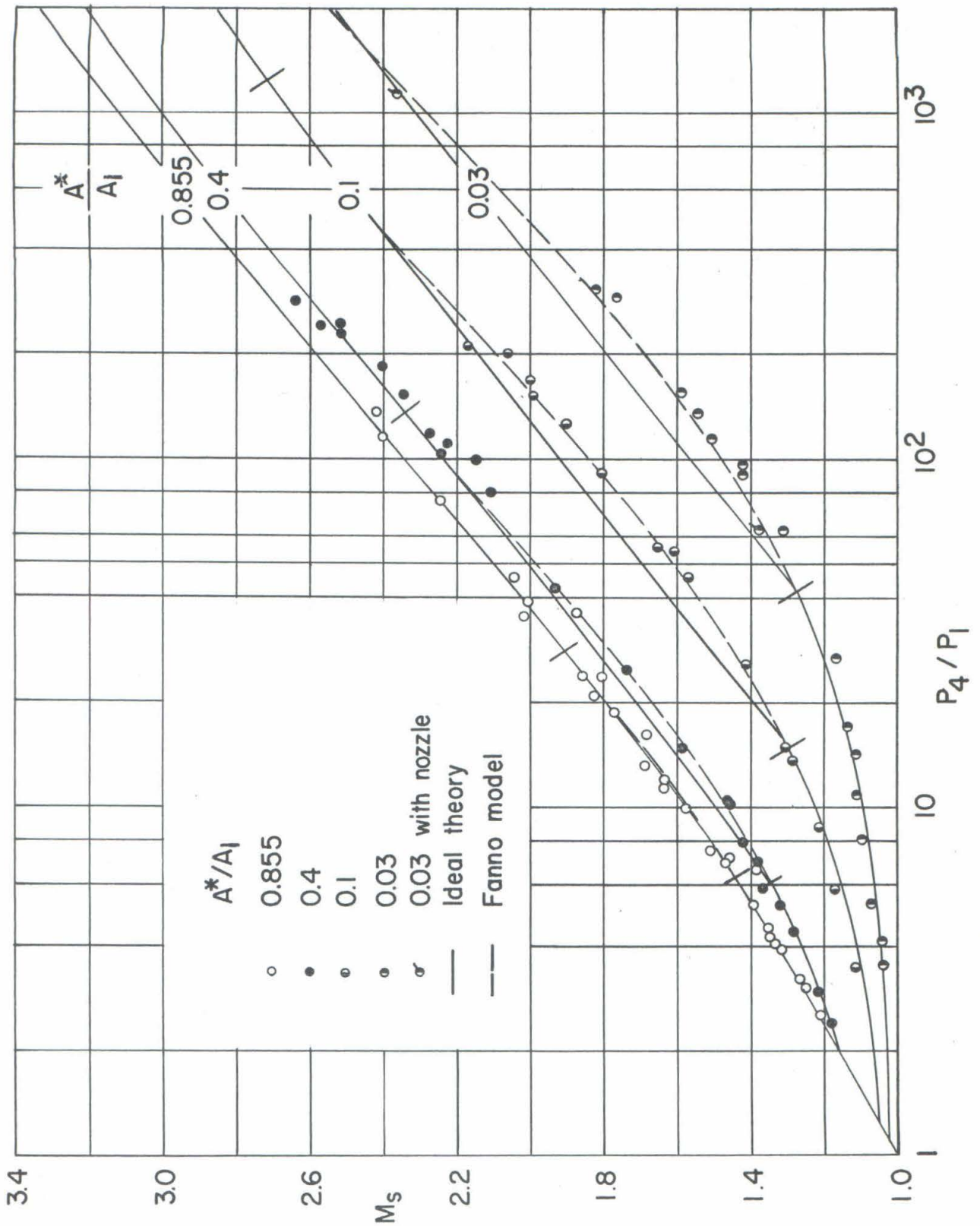
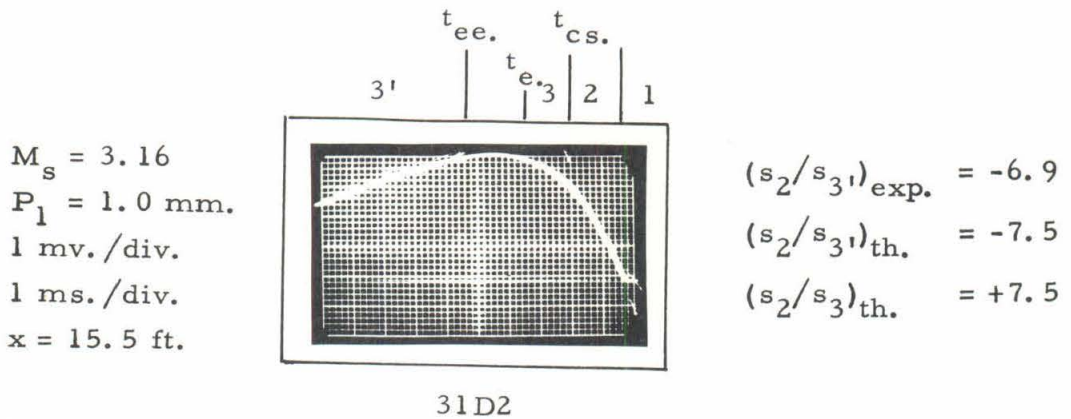
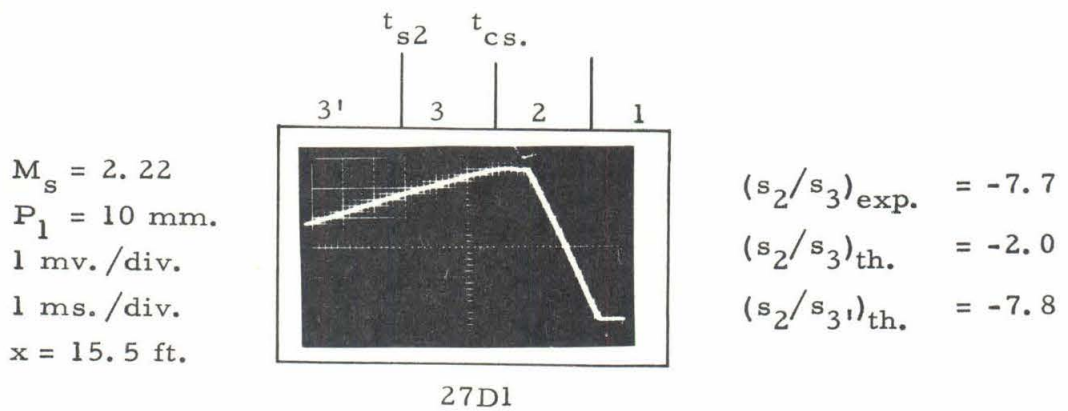


FIG. 9 CORRECTED DATA -- NITROGEN-AIR SHOCK TUBE
 $A_1/A_2 = 0.855$

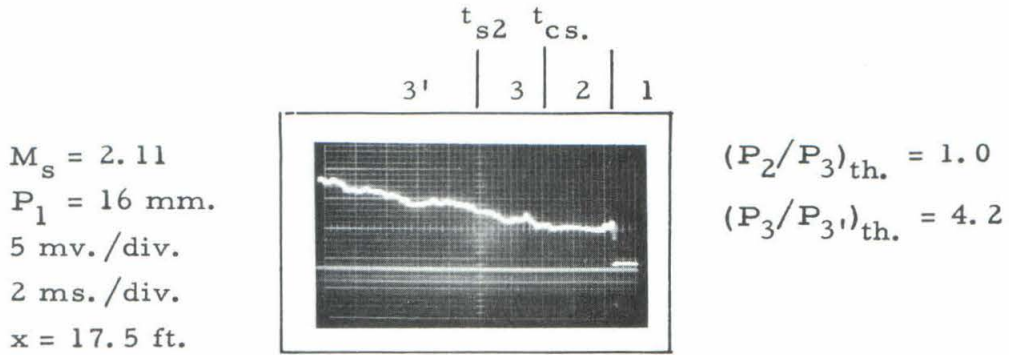


(a) Expansion Wave Configuration; $A^*/A_1 = 0.1$

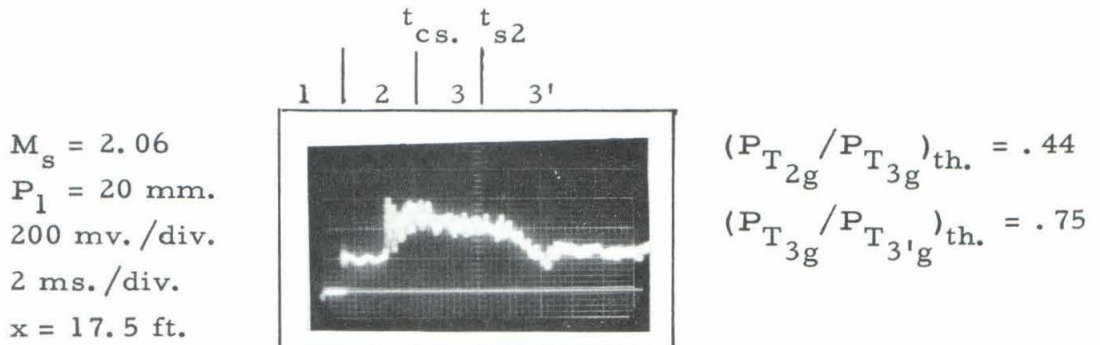


(b) Normal Shock Wave Configuration; $A^*/A_1 = 0.1$

FIG. 10 ILLUSTRATIVE FINE WIRE RESPONSE; $1\frac{1}{2}$ MIL. WIRE



5J6

(a) Side Wall Trace; $A^*/A_1 = 0.1$ 

3F2

(b) Total Pressure Trace; $A^*/A_1 = 0.1$

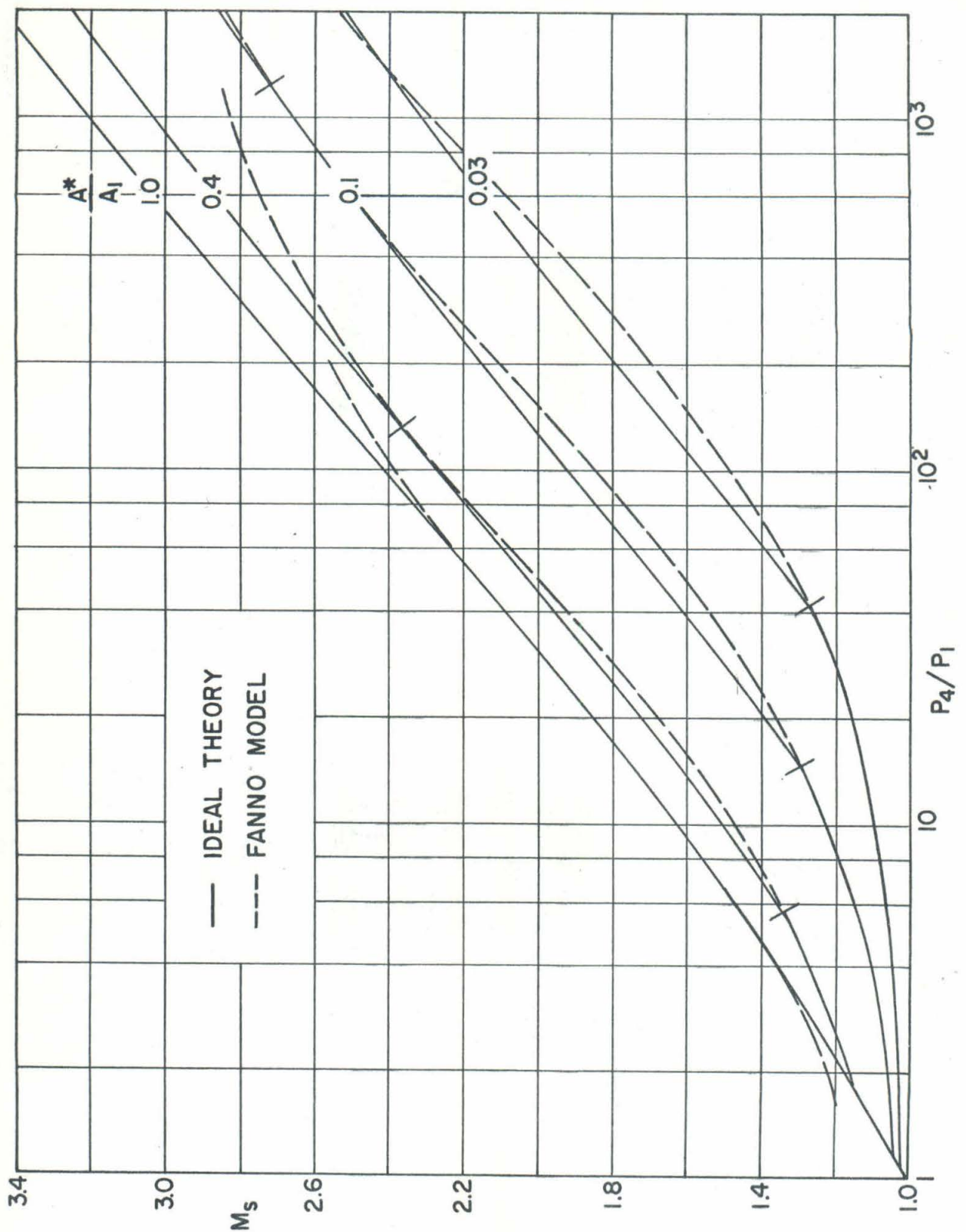


FIG. 12 THE FANNO PROCESS MODEL--NITROGEN-AIR SHOCK TUBE ;
 $A_4/A_1 = 1$

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